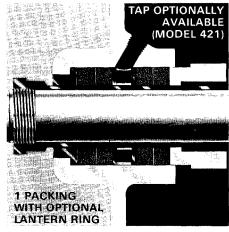
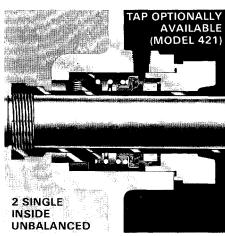
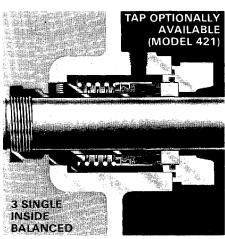
#### **ENGINEERING DATA** MECHANICAL SEALS AND PACKING







**PRESSURES** 

**TEMPERATURES** 

LIQUIDS



Standard packing on horizontal pumps and the standard mechanical seals on vertical pumps are suitable for most applications. Special sealing arrangements may however, be required due to higher pressure or temperature requirements and the nature of the liquid to be pumped. Factory option seals are of high quality and supplied by leading mechanical seal manufacturers. Various seal arrangements and types that better suit your specific needs are available. Seal faces are carbon vs. Ni-Resist on standard seals and carbon vs. Tungsten carbide on high temperature seals. Corrosion resistant alloy metal parts and Buna-N secondary sealing elements are provided. Various other metals are also available. Gland plates are cast iron and can be supplied in alternate materials. Recommendations and limitations are general. Specific selections can be offered only after rotating speeds, pressures, temperatures, type of equipment and liquid nature are known. The following illustrations describe the basic seal and packing options available. For options not shown refer to the factory. For quick reference for the type of seal best suited to your application, refer to the condensed information that heads each option. The following comments govern these recommendations:

1 PACKING Standard on Model 421. Not available on 422 and 423. PRESSURES (suction): Below atmospheric up to 250\* P.S.I.G. (Maximum pump limitation) A lantern ring is required on the first stage for suction lift applications.

TEMPERATURES: From minus 100°F. up to 275°F.\* with high temperature packing, or 225°F. with standard packing.

LIQUIDS: All liquids that are compatible with graphited fiber packing. Other packings are available for special applications.

2 SINGLE — UNBALANCED Standard on Model 422 and 423. Optional on Model 421.

PRESSSURES (suction): Below atmospheric up to 100 P.S.I.G.

TEMPERATURES: From minus 100°F, up to 275°F, with high temperature seals, or 225°F, with standarad seals.

LIQUIDS: All liquids that are compatible with the seal materials of construction and with a specific gravity higher than .6.

3 SINGLE — BALANCED Optional on all Models.

PRESSURES (suction): Up to 250 P.S.I.G. (Max. pump limit)

TEMPERATURES: Minus 100°F. up to 275°F, with high temperature seals, or 225°F. with standard seals. LIQUIDS: All that are compatible with the seal materials of construction and with a specific gravity of .6 or lower.

PRESSURES — The pressures referred to are those found at the pump suction. Most seal manufacturers recommend a flushing arrangement from the discharge to the stuffing box where "below atmospheric pressure" is encountered. The 420 Series first stage stuffing box incorporates an internal bypass arrangement which permits flushing to the mechanical seal. External bypasses are available to both seal faces. An external bypass is standard on vertical pumps to the upper seal face.

TEMPERATURES — The temperature limitation of a mechanical seal is frequently determined by the shaft sealing material. The various elastomer "O" ring materials have varying temperature limits, depending upon the chemical and/or physical properties of the process fluid. Filled. TEFLON†, shaft seal rings are available.

LIQUIDS — Due to varying degrees of resistance of various sealing compounds in different pumped liquids, the following mechanical seal sealing rings are available: BUNA-N, NEOPRENE, VITON, TEFLON† and other synthetic elastomers.

†DUPONT registered trademarks.

\*NOTE: Hardened stainless steel (450 minimum brinnel) shaft sleeves are available with this option and are required when the suction pressure is over 100 P.S.I.G. or when the temperature exceeds 225°F.

DATED NOVEMBER 2001

SUPERSEDES PAGE 72 DATED OCTOBER 1982

# ENGINEERING DATA INTERCHANGEABILITY AND POWER SERIES

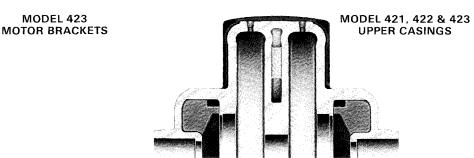
420 SERIES \_\_\_\_\_

Aurora Models 421, 422 and 423 were designed for maximum interchangeability. Each model is available in 9 different sizes, offering a model and size

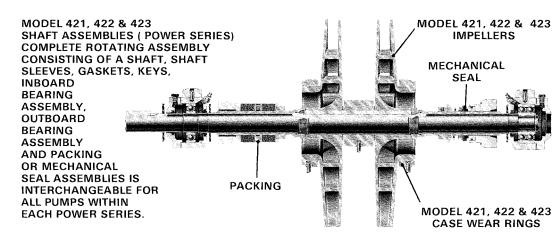
precisely fitted to the installation requirements. The 9 sizes are divided into 4 "power series." Within each power series, all parts are completely interchangeable except for the impeller, casing, and case wearing rings for either right hand or left hand rotation. See the illustration below for all details.

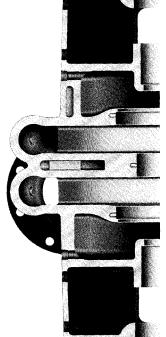


MODEL 422 & 423 LOWER CASINGS



MODEL 421 LOWER CASINGS



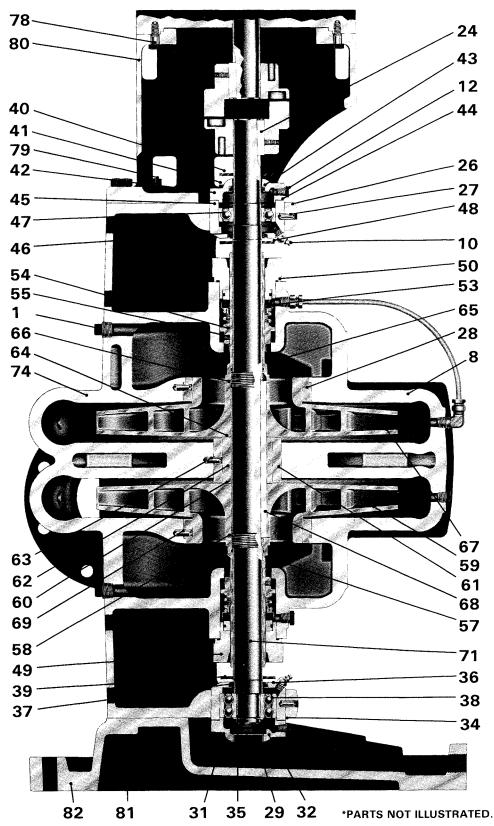




POWER SERIES	PUMP SIZE EXAMPLE: 3 x 4 x 14 (3-Discharge Dia.) (4-Suction Dia.) (14-Approx. Max. Impeller Dia.)												
2	3	4A	4	5A	5								
2 x 2-1/2 x 12A	3 x 4 x 14A	5 x 5 x 12*	5 x 6 x 15*	6 x 6 x 12*	6 x 8 17A*								
2 x 2-1/2 x 12B	3 x 4 x 14B				6 x 8 x 17B*								
2-1/2 x 3 x 12A	4 x 5 x 15*		*Model 421	Pumps Only									



### ENGINEERING DATA **MATERIALS OF CONSTRUCTION**



		_	PUMP CONSTRUCTION											
Pc No	Descrip. (*Not	Bronze	All	All Stain.										
1	Shown) Plug	Fitted Mall. Iron	Bronze Bronze	iron Mall, Iron	Steel									
2	*Plug	A197	Wrought	A197	Stain, Stl. AISI 316									
6	*Capscrew	Steel SAE 2	Steel SAE 2	Steel SAE 2	Stain, Stl. AISI 316									
8	*Capscrew Casing	Cast Iron	Bronze	Cast Iron	Stain. Stl.									
	Half	A48	B62-4A	A48	ACI CF8M									
10	*Gasket Gr. Ftg.	Ві	na-N Treat Steel		se									
12	Plug	Ma	lleable Iro											
18	*Nut	Bronze \	<b>V</b> rought	Steel SAE 2	Stain, Stl. AISI 316									
19	*Washer	Cad. Pit. Steel	Bronze Wrought	Cad. Plt. Steel	Stain. Stl. AISI 316									
20	*Gland Clp. *Gland	Cast Iron	Bronze	Cast Iron	Stain. Stl.									
22	*Swing	A48 Cad. Plt.	B62-4A Sil. Brz.	A48 Cad. Plt.	ACI CF8M Stain, Stl.									
	Bolt	Steel	Wrought	Steel	AISI 316									
23	*Packing Key		Graphite Steel W											
25	*Capscrew	Steel	Bronze	Steel	Stain, St1.									
26	Bearing	SAE 2 Cast Iron	Wrought Bronze	SAE 2 Cast Iron	AISI 316 Stain, Stl.									
	Сар	A48	B62-4A	A48	Stain. Stl. ACI CF8M									
27	Pin	Cad. PIt. Steel	Stain, Stl. AISI 416	Cad. Plt. Steel	Stain.Stl. AISI 316									
28	Case Ring	Bro ASTM E		Cast Iron A48	Stain, Stl. ACI CF8M									
29	Protector		Steel V	rought										
31	Capscrew Cart. Cap			SAE 2 ASTM A48										
34	Gasket	В	una-N Trea		se									
35	Ret. Ring		Spring Cast Iron											
36	Cartridge Gr. Seal			nd Steel										
38	Bearing		Steel Co											
39-4	O Slinger Capscrew	l	Neor Steel											
42	Cart. Cap		Cast Iron	ASTM A48										
43	Gr. Seal Gasket	D	Buna-N a una-N Trea	and Steel										
45	Cartridge			ASTM A48	136									
46	Gr. Seal			and Steel										
47	Bearing Slinger			mmercial orene										
49	Gland	Cast Iron A48	Bronze B62-4A	Cast Iron A48	Stain. Stl. ACI CF8M									
50	0-Ring	A48	Boz-4A Bun		ACI CROW									
52	*Lantern Ring	Bro ASTM I	nze R62.44	Cast Iron A48	Stain, Stl. ACI CF8M									
53	M. Seal		St. Stl. (2)	St. Stl. (1										
54	Collar		nze B62-4A	Cast Iron	Stain. Stl. AISI 316									
55	Setscrew	S	tainless St		16									
56	*Bushing		nze 862-4A	Cast Iron A48	Stain. Stl. AISI 316									
57	Sleeve	Bronze Hig	gh Lead Tin											
58 59	Gasket Impeller	Bronze	Bronze	DuPont Cast Iron	Stain. Stl.									
	Gasket	B119	B119 Teflon	A48	ACI CF8M									
60	Bushing	Bro	nze	Cast Iron	Stain. Sti.									
62	Sleeve	ASTM I		A48	ACI CF8M									
63	Pin	Cad. Plt. Steel	Stain. Stl. AIS1 416	Cad. Plt. Steel	Stain, Stl. AISI 316									
64 65	Gasket Sleeve	Bronzo Hic	Teflon th Lead Tin	DuPont Istain Sti	A151 316									
66	Gasket	Dionzerna	Teflon		. 7(0) 310									
67	Impeller	Bronze B584	Bronze B584	Cast Iron A48	Stain. Stl. ACI CF8M									
68	Key	Stain. Stl.	Stain, Stl. AISI 316	Stain, Stl.	Stain, Stl. AISI 316									
69	Pin	AISI 416 Cad. Plt.	Stain. Stl.	AISI 416 Cad. Plt.	Stain, St1.									
70	*lmp. Ring	Steel	AISI 416	Steel Cast Iron	AISI 316 Stain. Stl.									
		B5	84	A48	AISI 316									
71	Shaft	Stl. AISI C1045	Stain, Stl. AISI 316	Stl. AISI C1045	Stain, Stl. AISI 316									
72	*Pin	Cad. Plt.	Stain. Stl.	Cad. Plt.	Stain. Stl.									
73	*Pin Casing	Steel Cast Iron	AISI 416 Bronze	Steel Cast Iron	AISI 316 Stain. Stl.									
	Half	A48	B62-4A	A48	ACI CF8M									
75 76	*Drive Scr. *Name Plt.	S	Steel Bro tainless St	nze Plated eel AISI 3	03									
78-7	9 Capscrew		Steel	SAE 2										
80 81	Bracket Capscrew			ASTM A48 SAE 2										
82	Base		Cast Iron	ASTM A48										
All	material sp	ecifications	are in ac	cordance	with ASTM									

All material specifications are in accordance with ASIM unless otherwise noted. (1)  $B_{30}P_{66}171(JC)$  (2)  $XP_{66}1C1(JC)$ .



#### **ENGINEERING DATA**

10 EXTERNAL PIPING can be provided when it is necessary to filter and regulate the flow of liquid to the stuffing box. With this option, piping is provided from the pump discharge to both stuffing boxes. If the pumped liquid is not suitable for sealing purposes, the standard internal passages can be plugged and

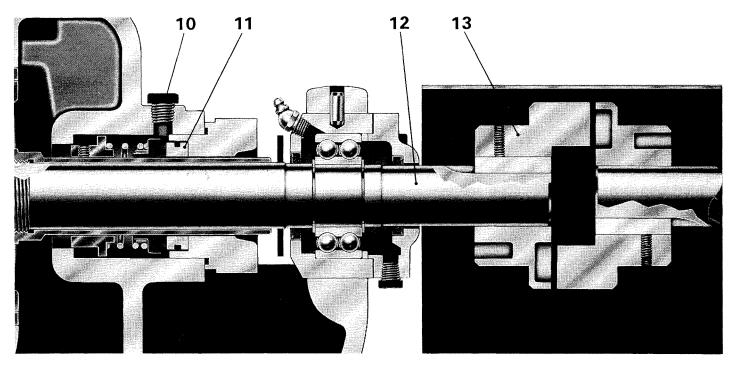
external piping from a water seal unit can be provided directly to the stuffing box or seal chamber. Lantern rings are required with this option on packed pumps.

11 MECHANICAL SEALS are available for special applications or hazardous service in single, balanced, and unbalanced designs. Packing with a lantern ring is available.

12 DOUBLE EXTENDED SHAFT option provides for dual drive appli-

cations such as an electric motor prime driver and stand-by diesel or internal combustion engine.

13 FLEXIBLE COUPLING is required between the pump and driver. It compensates for minor misalignment and reduces the transmission of vibration from the driver to the pump system. Clutch type couplings are available for the dual drive systems.



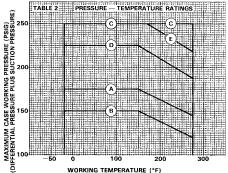
**★STANDARD PUMP:** Available in Bronze Fitted. Optional in All Bronze, All Iron, or Stainless Steel. Special materials are also available. 1. ME-CHANICAL SEALS. 2. LANTERN RINGS: Available for packed pumps only, provides lubrication under pressure to each stuffing box to extend packing life. An internal water seal passage provides the necessary lubricant from the pumped liquid. 3. FLUSHING LINES. 4. IMPELLER WEARING RINGS: Prevent rotational wear from occurring on the impeller and are easily replaced. The rings are press locked on the impeller. 5. CASE WEARING RINGS: Available in 316 Stainless Steel for longer life. 6. SHAFT SLEEVES: Minimum 450 Brinnel Hardened 440C Stainless Steel is recommended for abrasive applications on packed pumps only. Pumps with mechanical seals are available with 316 Stainless Steel sleeves. 7. SHAFT MATERIAL: Standard pumps do not require alloy shafts as Teflon sealed shaft sleeves protect the shaft from corrosion. On severe applications 316 Stainless Steel shafting is available. Alloy shaft is recommended when inside balanced seals are specified. 8. DOUBLE EXTENDED SHAFT. 9. VERTICAL PUMPS. OIL LUBRI-CATION: Recommended for special applications such as remote installations, etc. Available only in Model 421. 11. 250 PSI FLANGS: Suction and Discharge flanges machined to ASA flat face specifications. Special surface finishes such as raised face are available. 12. PETCOCK: Vents air manually from the upper casing during initial start up. 13. VENT TAPS: Oversize taps are available in either/or the upper casing or suction chambers. 14. BASES: Available in cast iron with drip rim, formed steel or structural steel. 15. ABRASIVE SEPARATORS: Available with option

3 to prevent entrained abrasives from entering the stuffing boxes via the recirculation or water seal liquid. 16. ORIFICE BY-PASS: Regulates a predetermined flow of liquid to the stuffing boxes where this is desired. 17. GLAND EYEBOLTS AND NUTS: For corrosive applications. Made of 316 Stainless Steel. 18. BRONZE PACKING GLANDS: For corrosive duty. 19. **ENGINEERING TESTS: Several tests** can be provided. (A) Certified Performance Test; (B) Certified Witness Performance Test; (C) Hydrostatic Test Submittal; (D) Vibration Test Submittal; (E) NPSH Test; (F) Witness NPSH Test. 20. COUPLING GUARD. 23. DOUBLE ROW IN-BOARD BEARING: Recommended for severe service such as direct drive with internal combustion engines. ADDITIONAL MODIFICATIONS are also available.



**ENGINEERING DATA** DATED OCTOBER, 1982

BEARING LIFE is based on the radial and thrust loads imposed	TABLE 1 PUMP CASING	MINIMU FOR STA AND DIS	PIPE SIZE	CODE	
on the bearings at the specific	MATERIAL	ANSI SPEC.	CLASSIFICATION	SILL	
operating head and suction			125 PSI Flat Face	1-12	Α
pressure. The Split case pump is	Cast Iron	B16.1	123 (3) (10)	14-24	В
designed for two year minimum	ASTM A48		250 PSI Flat Face	1-12 14-24	С
B <sub>10</sub> life at the maximum recom-	Bronze	B16.24	150 PSI Flat Face	All	D
mended loads. Bearing life at any	ASTM B62	B10.24	300 PS1 Flat Face	AII	С
other point of greater capacity on	Stainless Steel	B16.5	150 PSI Flat Face	All	E
the curves will greatly exceed the	ASTM A743 Grade CF8M	D10.5	300 PSI Flat Face		С
minimum life shown. Average bearing life is equal to five (5)	Maximum Hy working pres		Pressure 1½ times Pressure 1½ times	maximuı	m case
times the minimum bearing life (note*). SHAFT DEFLECTION is	TABLE	1121421411211121	SURE TEMPERATURE		



EXAMPLE: A model 420 Pump with a bronze casing has been selected for operating at a case working pressure of 240 P.S.I.G. at 150°F. Enter Table 2 at 150°F. and read upward to 240 P.S.I.G. it is determined that the selection is within the recommended maximum case working pressure area for 300 PSI flanges and is therefore acceptable. Note that the example exceeds the maximum case working pressure unit if the material selected would have been 125 PSI flanged cast iron or 150 PSI flanged bronze.

TABLE 3 SPECIFIC GRAVITY OF COMMON METALS											
TYPE METAL	CAST Bronze	CAST IRON	CARBON STEEL	STAIN. STEEL							
SP. GR.	8.86	7.20	7.84	7.90							

TABLE 4 († LESS IM		ABLE 9 UIET PU	MP DAT	TABLE 11 - PUMP Size Factor					
PUMP SIZE	POWER SERIES WR² ROT ELEMENT		MAX. IMP. DIA.	CUT WATER DIA	QUIET IMP. DIA.	SPHERE DIA	3500 RPM	1750 RPM	1150 RPM
2 x 2-1/2 x 12A			12.00	13.25	11.25	.25	.50	.65	-
2 x 2-1/2 x 12B	2	.025	12.00	13.25	11.25	.31	-	.70	-
2-1/2 x 3 x 12B			12.00	13.25	11.25	.25	.60	.65	.70
3 x 4 x 14A			14.00	15.50	13.25	.50	-	1.15	1.25
3 x 4 x 14B	3	.060	14.50	15.50	13.25	.43	-	1.10	-
4 x 5 x 15			15.00	16.53	14.00	.68	-	1.40	1.50
5 x 5 x 12	4A	RTF	12.00	13.13	12.00	.70	RTF	-	-
5 x 6 x 16	4	.099	15.00	16.56	14.00	.68	-	1.40	-
6 x 6 x 12	5A	RTF	12.00	13.13	12.00	.70	RTF	-	-
6 x 6 x 17A	5	.210	17.00	18.75	16.00	.68	-	1.80	-
6 x 8 x 17B			17.00	18.75	16.00	.81	-	1.65	1.75
1-1/2 x 3 x 9	2	.025	8.75	9.50	8.00	.25	.40	.45	-
2x4x9			8.62	8.53	8.00	.31	.45	.50	_

TABLE 12	CHART R.P.M.	DESIRED R.P.M.	MULTIPLY LIFE BY
SPEED	3500	1750	2
(R.P.M.)	3500	1150	3
FACTORS	1750	1150	1.5

TABLE 15		POWER	SERIES	
DIMENSION & DESCRIPTION	2	3	4	5
A STUFFING BOX I.D.	2.43	2.81	3.06	3.43
B STUFFING BOX DEPTH	3-1/8	3	3-1/2	3-3/4
C O.D. OF SLEEVE	1-1/2	1-3/4	2	2-3/8
PKG. RINGS W/O LANT. RING	12	10	12	12
PKG. RINGS W/LANT. RING	10	8	10	10
RING IN FRONT OF LANT.	2	2	2	2
PACKING SIZE (SQ.)	7/16	1/2	1/2	1/2
D WIDTH OF LANT, RING	5/8	5/8	3/4	3/4
E NEAREST OBSTRUCTION	1-5/8	1-3/4	1-3/4	2
F DIA, OF MECH, SFAL SEAT	2-1/8	2.1/2	2-3/4	3-1/4
G LENGTH OF MECH. SEAL	1-9/16	1-7/8	2	2-3/8
J SHAFT DIA. AT IMPELLER	1-3/8	1-5/8	1.7/8	2-1/8
K SHAFT DIA. AT SLEEVE	1-1/4	1-1/2	1.3/4	2
L SHAFT DIA. AT CLPG. END	1-1/8	1-3/8	1-1/2	1-3/4
MAX. DEFL. @ SEAL FACE	.002	.002	.002	.002
INBOARD BEARING NO.	206	207	208	309
OUTBOARD BEARING NO.	5305	5306	5307	5309
M BEARING CENTERS	20 3/4	24-1/2	27-3/8	30
MIN. BEARING LIFE"	6 YR.	6 YR.	6 YR.	6 YR.

MAXIMUM CASE WORKING PRESSURE is the sum of the differential pressure and the suction pressure. Table 2 indicates the maximum case working pressure for the 420 Series Split Case Pumps in various materials and at various operating temperatures. These maximum allowable pressures are based on wall thickness for the particular series of pumps, ratings for American Standard Flange Specifications, see Table 1, and take into account the material at various allowable application temperatures. EXTERNAL INERTIA OR FLY-WHEEL EFFECT is the Kinetic energy stored in the rotating assembly that must be overcome when the pump impeller is caused to rotate within the casing. This energy frequently must be calculated to determine the torque required to start, accelerate or decelerate the pump. If the acceleration is rapid. the torque may be several times greater than the torque required to run the pump at normal or constant speed. WR2 values in LBS-FT<sup>2</sup> are provided for these calculations. See tables 3 thru 6.

WR2 values	given in tables are for bronze	
impeller		LB-FT <sup>2</sup>

**EXAMPLE 1:** Find WR<sup>2</sup> value for a 15" diameter 5 x 6 x 15 bronze fitted pump handling cold water. From chart the "WET" value for

a 15" diameter impeller . . . . . Add power series 4 rotating element

.09 LB-FT2 Total 16.69 LB-FT2

**EXAMPLE 2:** Find WR<sup>2</sup> value for a 15" diameter 5 x 6 x 15 all iron pump handling 0.67 specific gravity gasoline.

rom chart select "DRY" value and correct for difference in materials.

Sp. Gr. cast iron × 14.9 LB-FT<sup>2</sup> . . . . 12.09 LB-FT<sup>2</sup>

Take difference ("WET"-"DRY") values and coror difference in specific gravities.

1.70 x 0.67 . . . ..... 1.14 LB-FT<sup>2</sup>

Add power series 4 rotating element

less impeller .09 LB-FT2 Total 13.32 LB-FT<sup>2</sup>

pressure. PROCEDURE FOR DETERMINING MAXIMUM SHAFT DEFLECTION AND MINIMUM BEARING LIFE.

the consequence of the

unbalanced hydraulic force

acting inside the pump on the impeller and shaft in a radial

direction. This unbalance occurs

when the pump is operating away

from its best efficiency point. At

shut-off condition (zero flow) the

unbalance is greatest and

therefore the resultant radial load

is maximum. Radial load and

shaft deflection approach zero at

the best efficiency point of the

pump. 420 Series pumps are de-

signed for a maximum deflection of .002" at the mechanical seal

faces when operating at the maximum recommended differential

420 SERIES .

Determine the proper Pump Size, approximate Shut-Off Head in feet Power Series number, and Speed from the range charts illustrated on page 3 of 420 Bulletin

2 From table 11 determine the Pump Size Factor based on Pump Size and R P M

3 On table 13, page 76, locate the correct Shut-Off Head in feet and read across to the proper Pump Size Factor and down to the applicable Power Series Note the Load Factor in the process. Read to the scale on the left for the maximum Shaft Deflection value

4 From table 14, page 76, using the Load Factor from step 3 above read across to the correct Power Series number and down for the min Bearing Life in hours

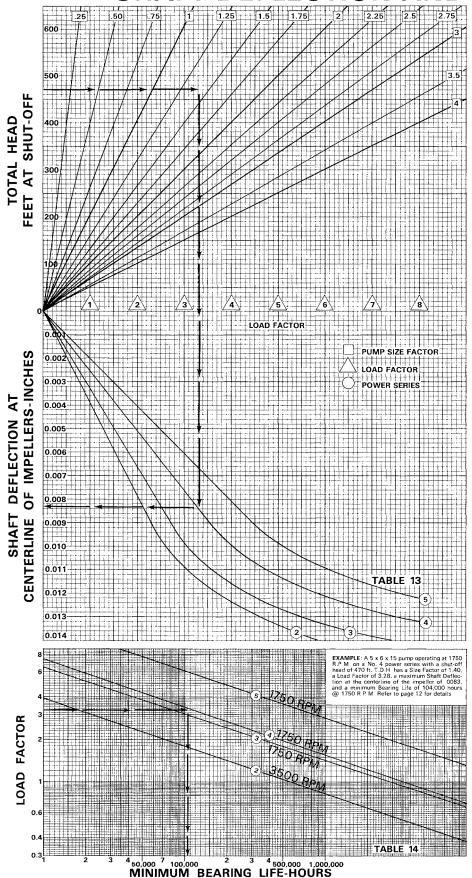
NOTE: 1. One (1) year life is based on 8740 HOURS (continuous operation) 2. Additional bearing information can be found on page 76 3. Specific information on Bearing Life and shaft Deflection can be obtained from the factory

TABLE		MODI	EL 430		MODEL 420														MOD	EL 420							
5	1-1/2 x	3 x 9	2 x	4 x 9	2 x 2-1/	2 x 12A	x 12A 2 x 2/1/2 x 12B 2-/1/2 x 3 x 12A		6	3 x 4 x 14A		3 x 4	3 x 4 x 14B		4 x 5 x 15		5 x 5 x 12		x 15	6 x 6 x 12		6 x 8 x 17A		6 x 8 x 17B			
DIA	DRY	WET	DRY	WET	DRY	WET	DRY	WET	DRY	WET	DIA	DRY	WET	DRY	WET	DRY	WET	DRY	WET	DRY	WET	DRY	WET	DRY	WET	DRY	WET
12.0	-	-	-	-	4.96	5.19	4.69	4.99	4.53	4.88	17.0	-	-	-	-	-	-	RTF	RTF	-	-	RTF	RTF	26.6	30.3	25.0	28.5
11.5	-	-	-	-	4.29	4.45	3.95	4.18	3.65	3.92	16.5	-	-	-	-	-	-	RTF	RTF	-	-	RTF	RTF	22.6	25.5	21.6	24.5
11.0	1	-	ı	-	3.67	3.75	3.43	3.65	2.98	3.20	16.0	-	1	-	1	-	1	RTF	RTF	ı	-	RTF	RTF	20.0	22.6	19.9	22.6
10.5	-	-	-	-	2.97	3.04	2.91	3.10	2.42	2.61	15.5	-	-	-	-	-	-	RTF	RTF	-	-	RTF	RTF	17.8	20.1	18.4	20.9
10.0	-	-	-	-	2.52	2.61	2.44	2.58	2.02	2.19	15.0	-	-	-	-	14.7	16.4	RTF	RTF	14.9	16.6	RTF	RTF	15.8	17.9	17.0	19.3
9.5	-	-	-	-	2.08	2.16	1.94	2.06	1.66	1.78	14.5	-	-	11.3	12.3	12.5	13.8	RTF	RTF	13.5	15.1	-	-	14.5	16.2	15.7	17.8
9.0	-	-	-	-	1.75	1.80	-	-	1.41	1.52	14.0	10.2	11.1	10.2	11.2	10.8	11.9	RTF	RTF	12.2	13.6	-	-	13.1	14.7	14.5	16.4
8.0	0.86	0.89	1.02	1.08	1.18	1.22	-	-	0.99	1.06	12.0	5.65	6.10	5.80	6.30	5.80	6.25	-	-	7.24	8.05	-	-	-	-	9.85	11.0
7.0	0.58	0.60	0.59	0.64	0.79	0.81	-	-	0.71	0.76	11.0	4.00	4.34	4.05	4.40	-	-	-	-	5.55	6.15	-	-	-	-	7.30	8.20
6.0	0.41	0.43	0.28	0.31	0.52	0.54	-	-	0.48	0.51	10.0	2.72	2.94	-	-	-	-	-	-	-	-	-	-	-	-	-	
6.0	0.31	0.32	0.11	0.13	-	-	-	-	-	_	9.0	1.67	1.85	_	_	-	-	-	-	-	-	-	-	-	-	-	-
WGT.	20	)#	18	#	35	#	34	#	33	#	WGT	56	6#	56	#	67	7#	40	)#	7:	2#	4	2#	10	00#	98	B#



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## 1982 ENGINEERING DATA SHAFT DEFLECTION AND BEARING LIFE



QUIET PUMP operation is always desirable and sometimes essential. One of the most important factors for noise control in a pumping installation is the correct selection of a pumping unit for the system. To insure that the pump will run quietly. it should be selected so that it will operate as close as possible to the best efficiency point. At this point the hydraulic shock within the pump is at a minimum since the flow angle of the fluid from the tip of the impeller is correct for the casing design. Every pump is designed for the best efficiency point and operation at any other point on the characteristic curves is a compromise. The amount of turbulence on either side of the best efficiency point increases as the point of operation is moved along the curve from the maximum efficiency. Therefore, the greater the turbulence, the greater the noise generated. Hydraulic shock is also a factor if the periphery of the impeller passes too close to the cutwater. If the ratio of the impeller diameter to the cutwater diameter in centrifugal pumps is greater than 0.92, the pump is likely to be hydraulically noisy. In such instances the hydraulic pulses are actually differential pressures that occur when the impeller vanes pass the cutwater. Cutwater ratios of 0.9 to 9.5 are typical; however, significantly lower noise levels are achieved in pumps designed with a ratio of 0.7 to 0.75. Although there is an optimum gap for pump efficiency, increases of only 3%-5% may be realized by using the optimum. A cutwater ratio of 0.85 is commonly specified by practicing engineers, thereby realizing a minimum reduction in pump efficiency with a mean reduction in noise level. Table 9 offers recommended quiet impeller diameter at 85% cutwater ratio. See Aurora Pump for details.

The charts reflect the *worst possible* conditions at pump shut-off. The effect from the impeller, shaft sleeves, wearing rings and packing will *reduce* the amount of *deflection*.

